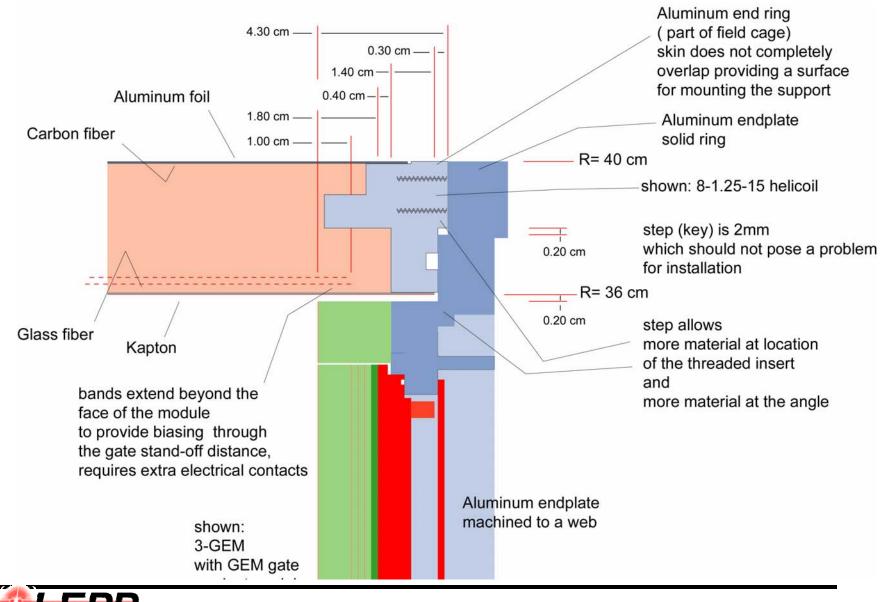
D. P. Peterson Cornell University, Laboratory for Accelerator-based ScienceS and Education

See also: http://w4.lns.cornell.edu/~dpp/linear\_collider/LargePrototype.html

This project is supported by the US National Science Foundation (LEPP cooperative agreement) and an LCDRD consortium grant

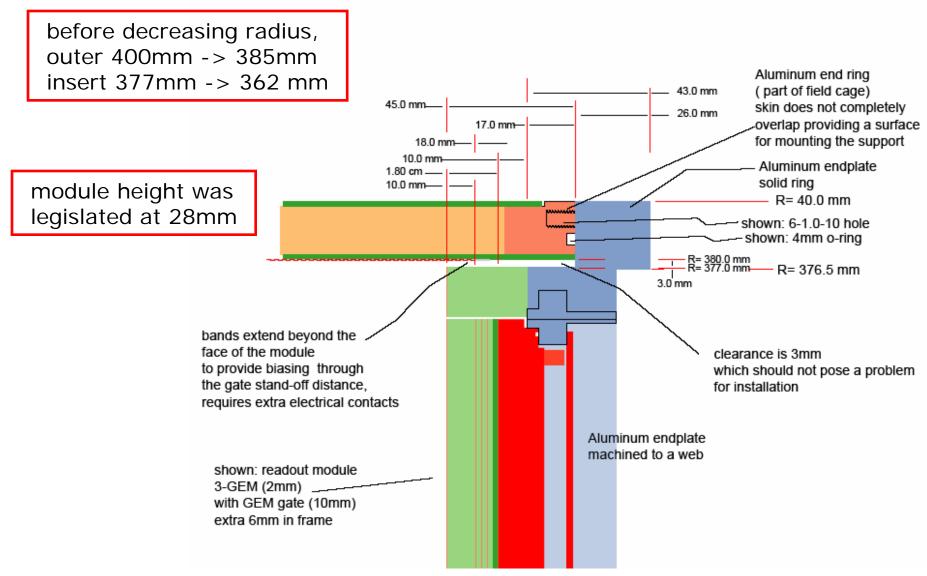


# Endplate/band geometry, 2006-11-09



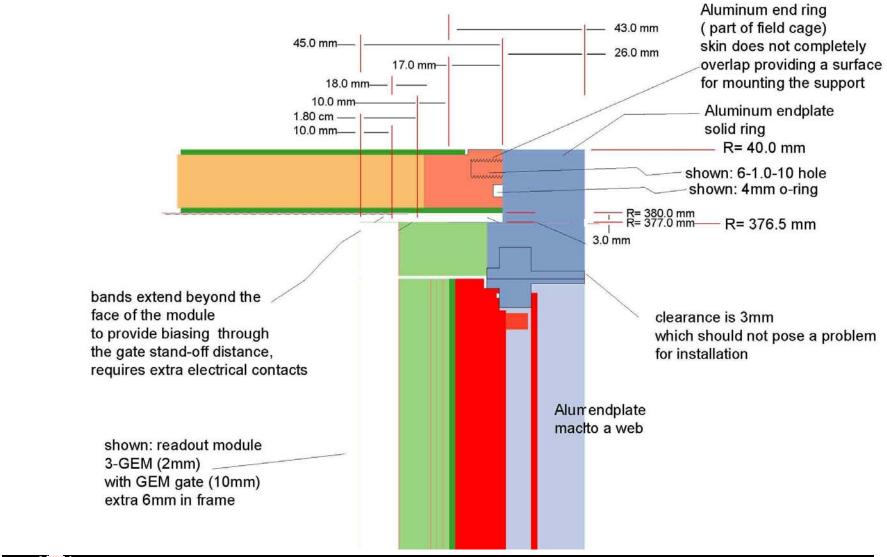


## Endplate/band geometry, 2007-01-17



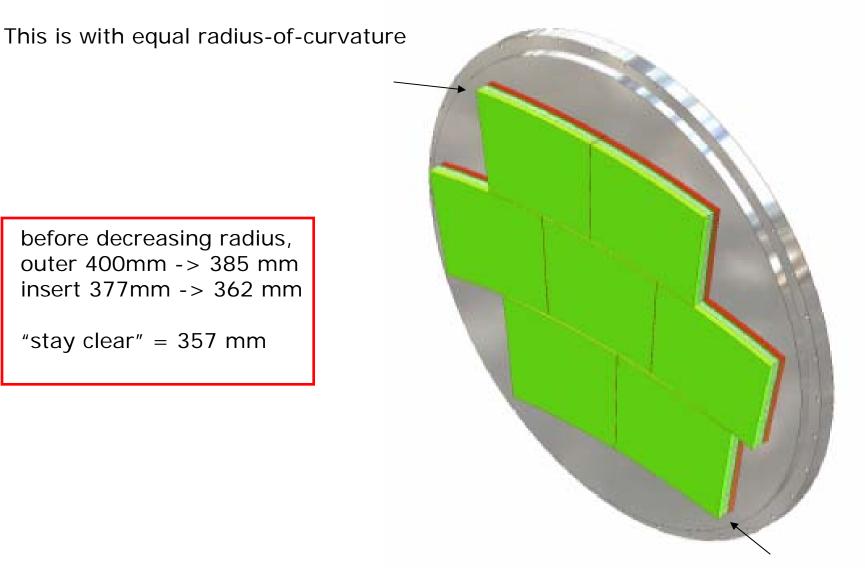


#### Endplate/band geometry, From Peter Schade 2007-05-23





## Endplate/Module model 2007-03-19



before decreasing radius, outer 400mm -> 385 mm insert 377mm -> 362 mm

"stay clear" = 357 mm



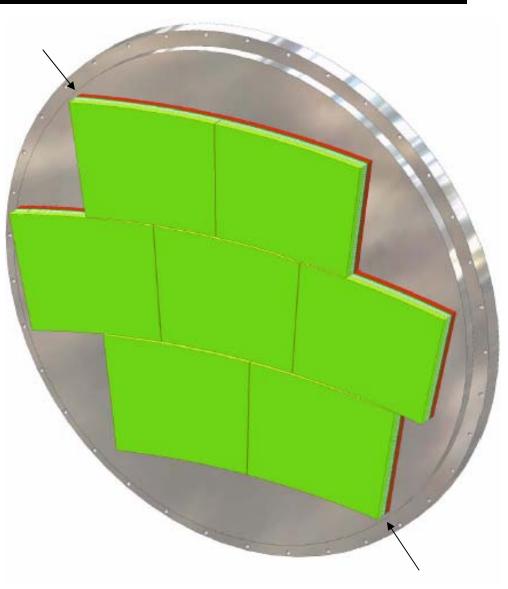
## Endplate/Module model, 2007-05-24

This is with equal center-of-curvature

after decreasing radius, outer 400mm -> 385 mm insert 377mm -> 362 mm

"stay clear" = 357 mm

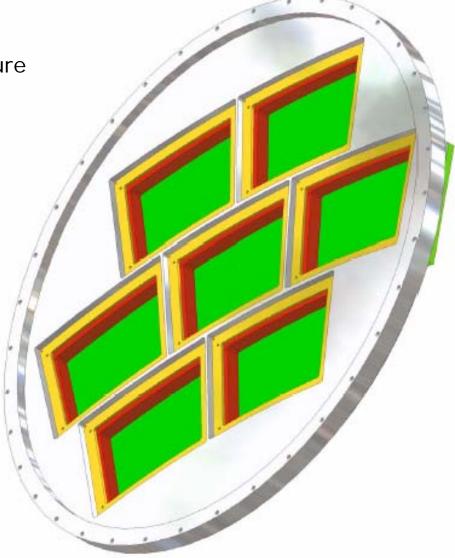
discuss the "bounding box"





## Endplate/Module model, 2007-05-24

This is with the equal center-of-curvature (outside)





# Endplate/Module model, 2007-05-25 (variation)

This is with not-equal center-of-curvature, not-equal radius-of-curvature.

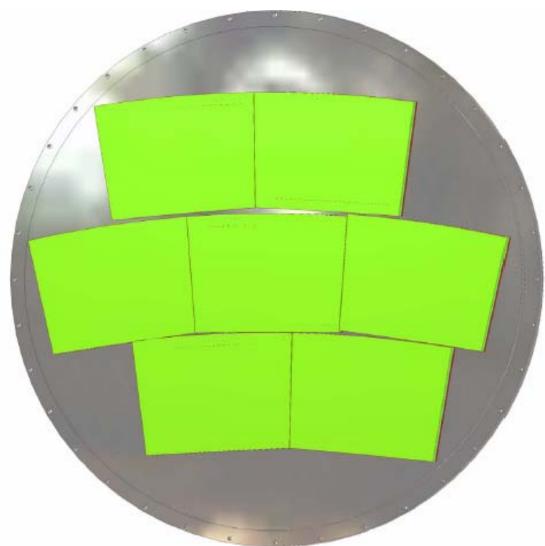
The lower radius of curvature is set to be almost flat, 9144 cm (100 yards).

The spread-sheet-driven model works. However, the extreme change in radius uncovers an error in the model.

This is an error in the way that the constraints are applied to the module positions.

This will be fixed.

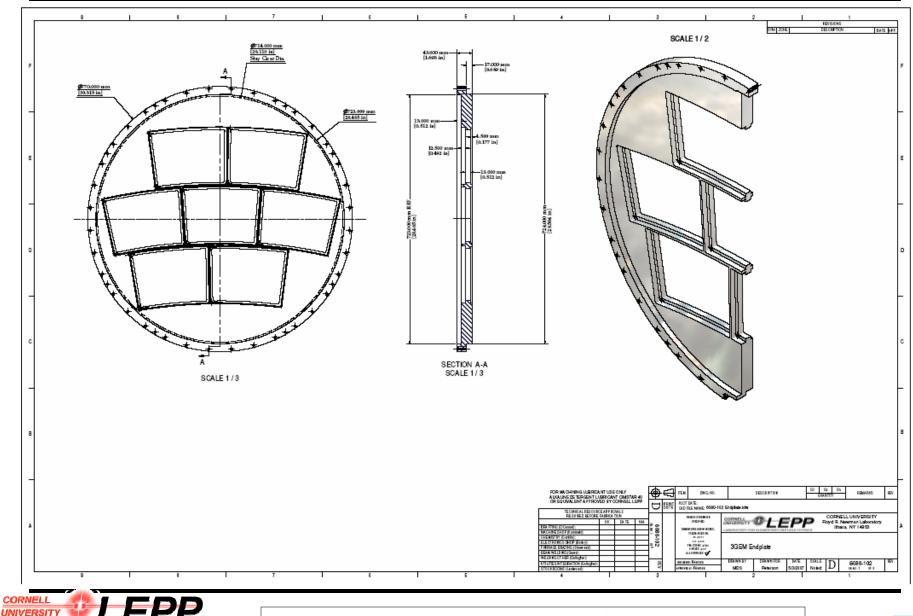
This does not affect the bounding specifications.





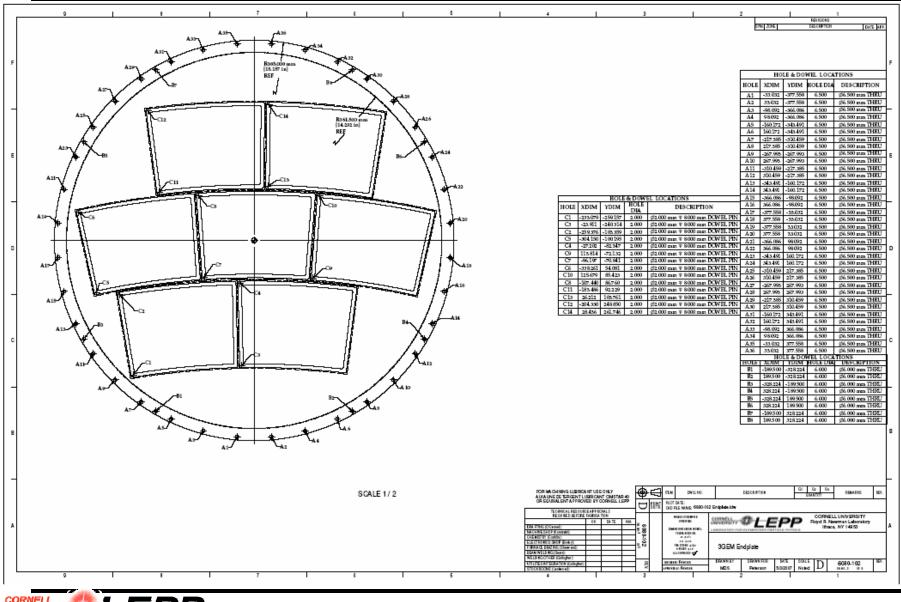
## Endplate Drawings, 2007-05-25

LABORATORY FOR ELEMENTARY-PARTICLE PHYSICS



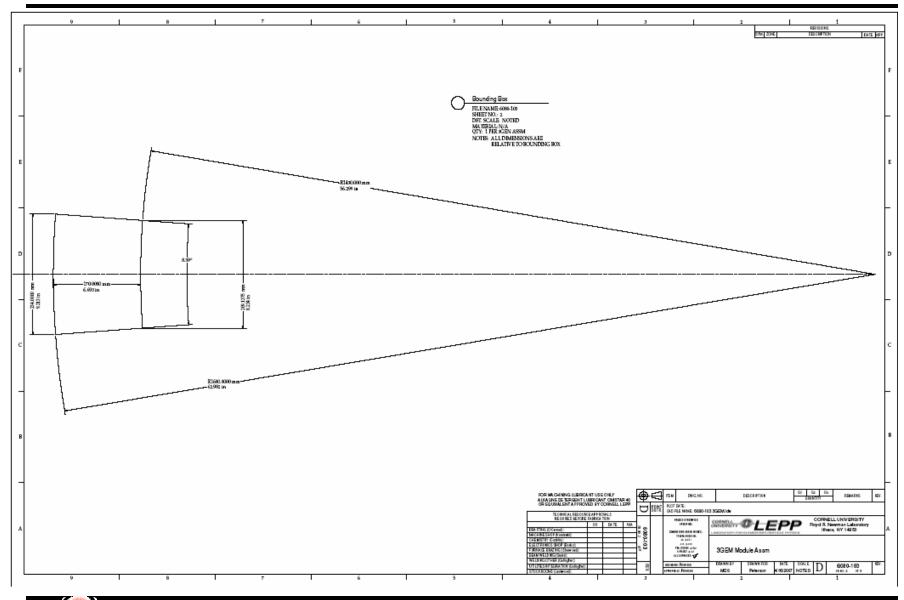
D. Peterson, "LC-TPC LP endplate", LC-TPC group meeting at LCWS07, 04-June-2007

#### Endplate Drawings, 2007-05-25

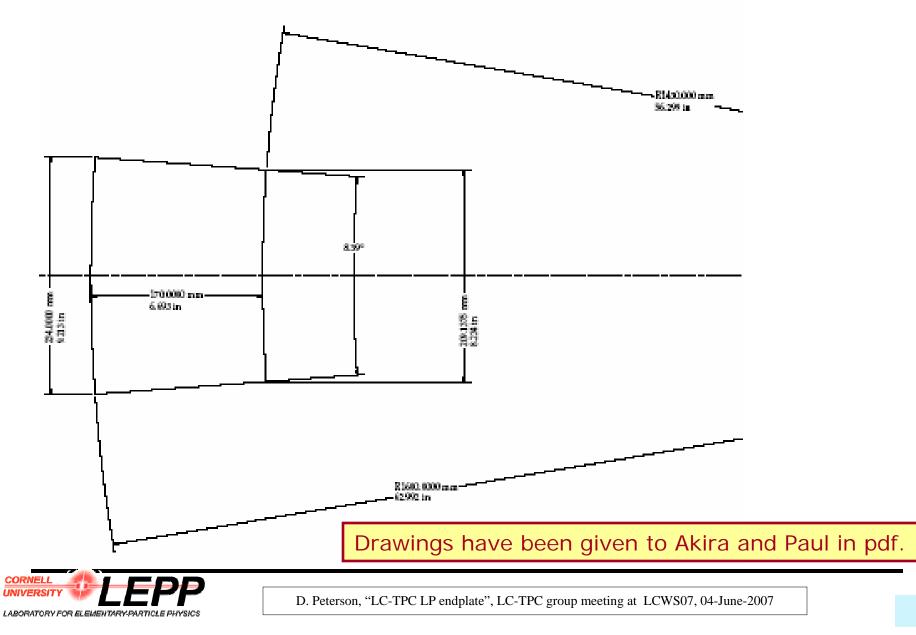




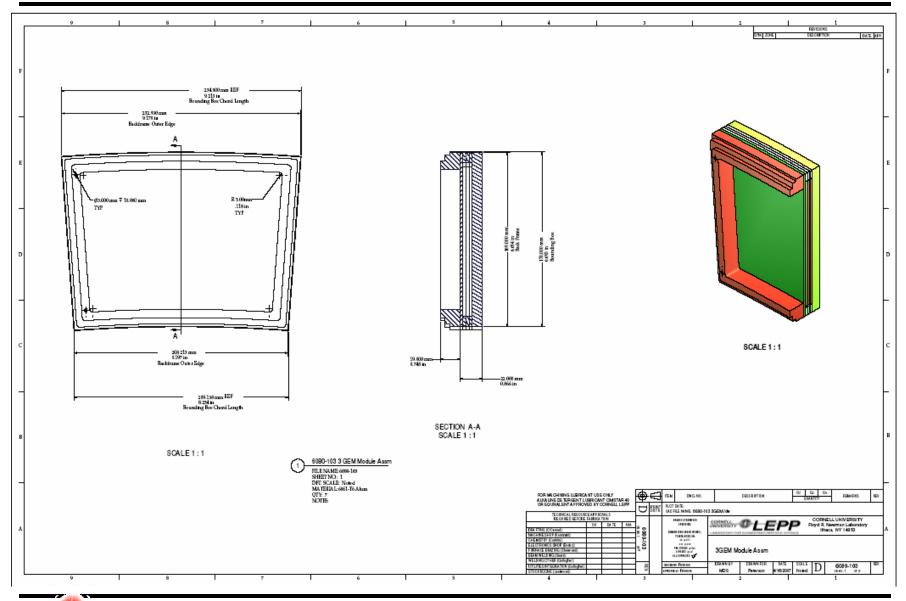
#### The Bounding Box, 2007-05-25





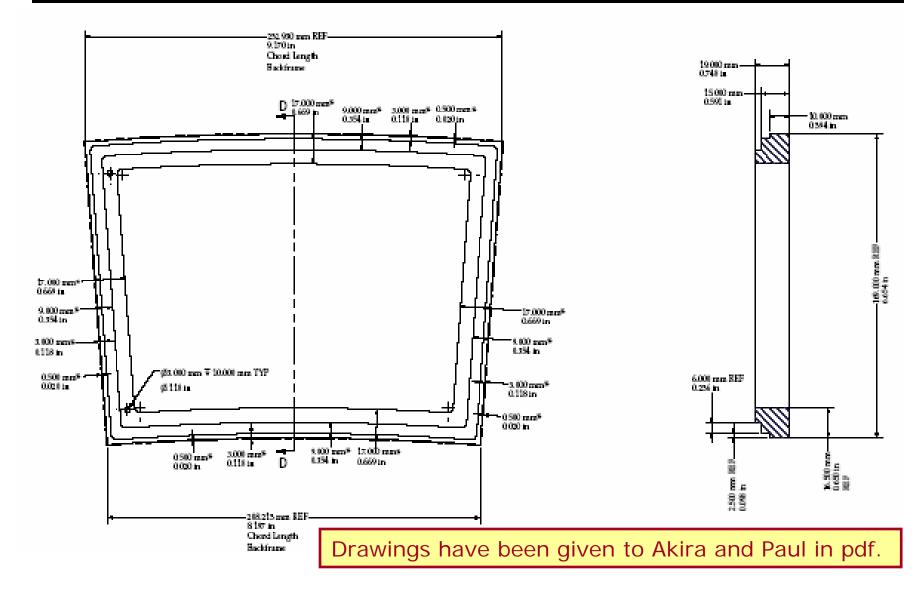


#### Module Drawings, 2007-05-25





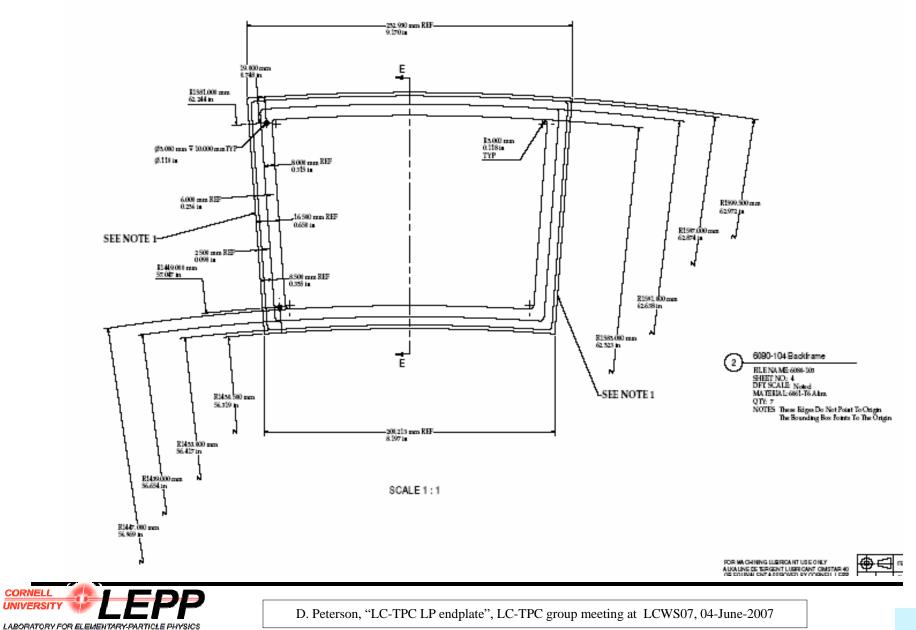
## Module Drawings, 2007-05-25





## Module Drawings, 2007-05-25

CORNELL



#### Stress relief test piece

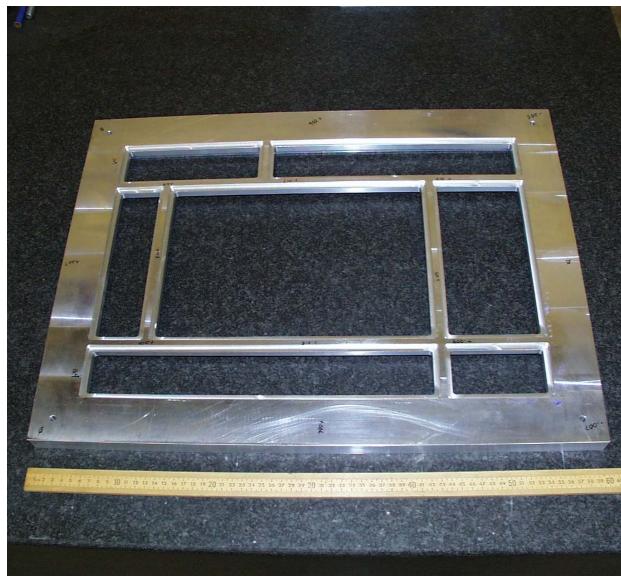
This shows the first in a series of "stress relief test pieces".

This has been cut with a center opening of 30cm wide. The "mullions" are the same size as proposed in the endplate drawing: 18mm at the widest width, 14mm in depth.

This is the first baseline part, with no stress relief.

It has been fully measured on a CMM. The mullion position is distorted upward by 500µm (0.020inch).

The drawing was modified to have the strengthening section as shown in the current endplate.





A close-up of the part shown in the previous slide.





# Machining a Stress Relief Test Piece, 2007-05-25

#### Motivation:

A position tolerance of  $<25\mu$ m is needed for the modules to decouple the calibration of the magnetic field from the position calibration of the modules.

I am trying to provide, at delivery,

<25µm position tolerance of the mullions. The endplate will then be evaluated after some service time to determine the ability to maintain this tolerance.

#### The program:

6 plates are being made to the revised drawing. A multi-step production is used:

- 1) machine to 1000  $\mu m$  oversize
- 2) machine to 750  $\mu$ m oversize,
- 3) stress relief
- 4) machine to 250 µm oversize,
- 5) stress relief
- 6) machine to drawing dimensions



hs. er ity to
drawing.
Stress relief processes: 2 plates - (3)heat to 325F, (5)heat to 650F

WARNING!

- 2 plates rapid cooling to liquid  $N_2$
- 2 plates ultrasonic cleaner, 6 hours

#### Coordinate Measuring machine (CMM), 2007-05-25





#### CMM, 2007-05-25, Z measurements

Example of measurement after the 2<sup>nd</sup> machining.

Units are milli-inch. 0.001 inch = 25.5  $\mu$ m

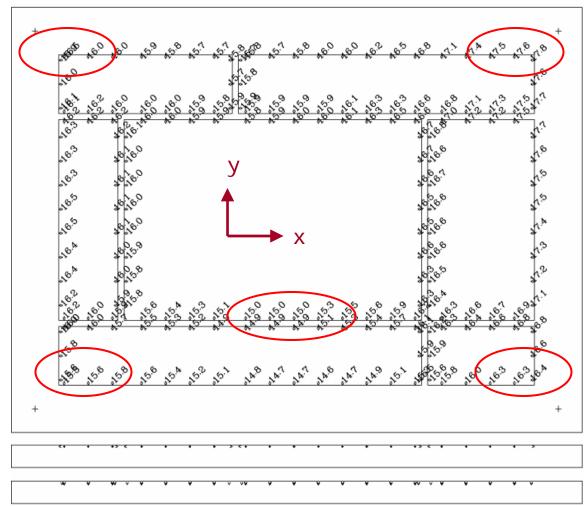
This is the Z view.

There is a 30  $\mu$ m bowing in z-x .

There is a twist about x from left to right of 25 μm.



 $\mathbf{Z}$ 





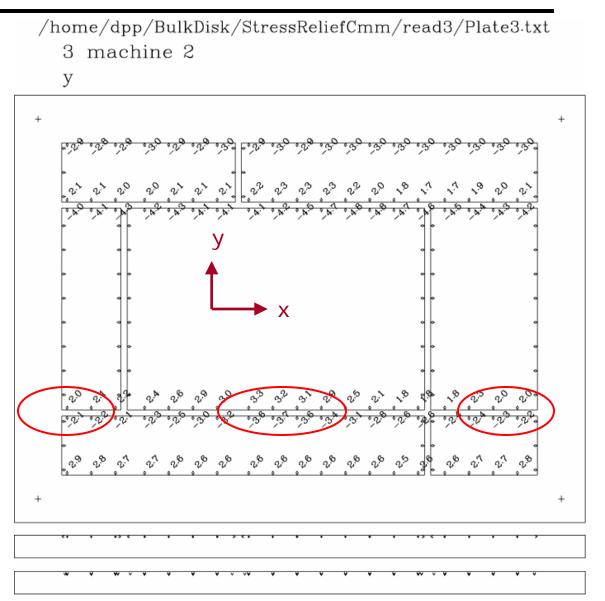
## CMM, 2007-05-25, y measurements

Example of measurement after the 2<sup>nd</sup> machining.

Units are milli-inch. 0.001 inch = 25.5  $\mu$ m

This is the y view.

There is a 30  $\mu$ m bowing in y of the indicated mullion.



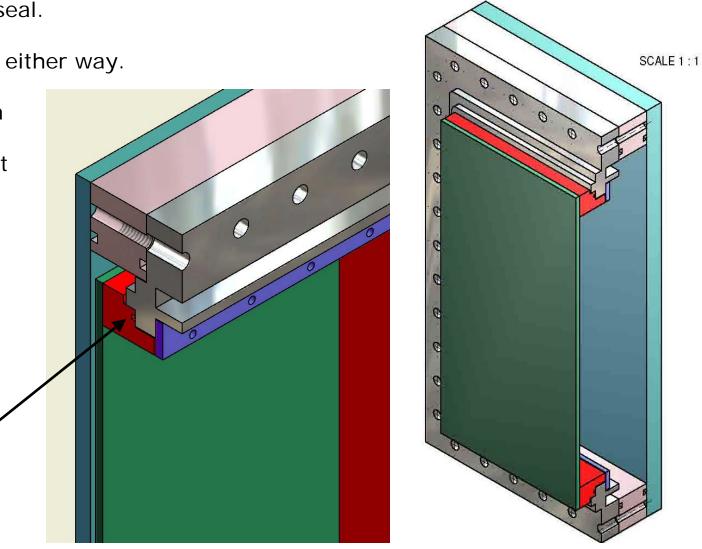


Gas Seal test, 2007-05-25

Test of the o-ring seal.

It can be mounted either way.

- model of mullion
- clamping bracket





#### Gas Seal test

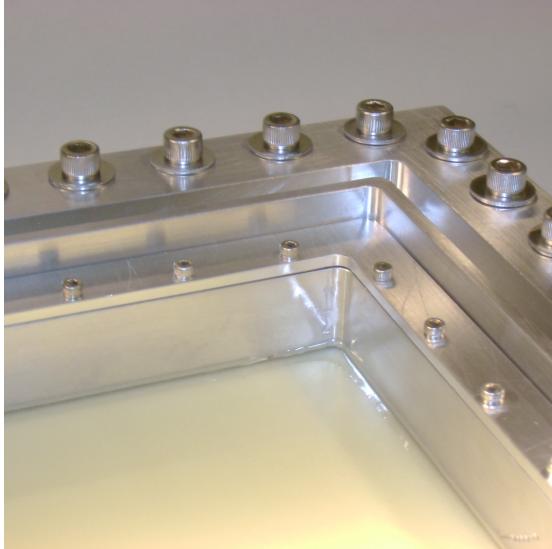


First test, atoxygen overpressure,~ 4 cc/hour/panel.

Second test (new back frame with improved o-ring slot) ~ 2 cc/hour/panel

Other improvements to make...





Interface of endplate to field cage

Bolt locations

